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| FORM PTO-1300 (REV 11-98) | | U.S. DEPARTMENT OF COMMERCE PATENT AND TRADEMARK OFFICE | ATTORNEY'S DOCKET NUMBER |
| TRANSMITTAL LETTER TO THE UNITED STATES DESIGNATED/ELECTED OFFICE (DO/EO/US) CONCERNING A FILING UNDER 35 U.S.C. 371 | | 4223/PCT | |
| INTERNATIONAL APPLICATION NO PCT/EP00/05210 ✓ | | INTERNATIONAL FILING DATE 07. June 2000 (07.06.00) | US APPLICATION NO. (if known) <i>107019336</i> (UNKNOWN - Be assigned) |
| TITLE OF INVENTION Electromagnetic Actuator and Method for Adjusting Said Electromagnetic Actuator | | | |
| APPLICANT(S) FOR DO/EO/US <u>Alexander VON GAISBERG; Dirk STRUBEL</u> | | | |
| Applicant herewith submits to the United States Designated/Elected Office (DO/EO/US) the following items and other information: | | | |
| <p>1. <input checked="" type="checkbox"/> This is a FIRST submission of items concerning a filing under 35 U.S.C. 371.</p> <p>2. <input type="checkbox"/> This is a SECOND or SUBSEQUENT submission of items concerning a filing under 35 U.S.C. 371.</p> <p>3. <input checked="" type="checkbox"/> This express request to begin national examination procedures (35 U.S.C. 371(l)) at any time rather than delay examination until the expiration of the applicable time limit set in 35 U.S.C. 371(b) and PCT Articles 22 and 39(1).</p> <p>4. <input checked="" type="checkbox"/> A proper Demand for International Preliminary Examination was made by the 19th month from the earliest claimed priority date.</p> <p>5. <input checked="" type="checkbox"/> A copy of the International Application as filed (35 U.S.C. 371(c)(2)).</p> <ol style="list-style-type: none"> <input type="checkbox"/> is transmitted herewith (required only if not transmitted by the International Bureau). <input checked="" type="checkbox"/> has been transmitted by the International Bureau. <input type="checkbox"/> is not required, as the application was filed in the United States Receiving Office (RO/US). <p>6. <input checked="" type="checkbox"/> A translation of the International Application into English (35 U.S.C. 371(c)(2)) with Translator's Declaration.</p> <p>7. <input checked="" type="checkbox"/> Amendments to the claims of the International Application under PCT Article 19 (35 U.S.C. 371(c)(3)).</p> <ol style="list-style-type: none"> <input type="checkbox"/> are transmitted herewith (required only if not transmitted by the International Bureau). <input type="checkbox"/> have been transmitted by the International Bureau. <input type="checkbox"/> have not been made; however, the time limit for making such amendments has NOT expired. <input checked="" type="checkbox"/> have not been made and will not be made. <p>8. <input type="checkbox"/> A translation of the amendments to the claims under PCT Article 19 (35 U.S.C. 371(c)(3)).</p> <p>9. <input checked="" type="checkbox"/> An oath or declaration of the inventor(s) (35 U.S.C. 371(c)(4)).</p> <p>10. <input type="checkbox"/> A translation of the annexes to the International Preliminary Examination Report under PCT Article 36 (35 U.S.C. 371(c)(5)).</p> | | | |
| Items 11. to 16. below concern document(s) or information included: | | | |
| <p>11. <input checked="" type="checkbox"/> An Information Disclosure Statement under 37 CFR 1.97 and 1.98, Form PTO-1449, 4 reference(s).</p> <p>12. <input type="checkbox"/> An assignment document for recording. A separate cover sheet in compliance with 37 CFR 3.28 and 3.31 is included.</p> <p>13. <input checked="" type="checkbox"/> A FIRST preliminary amendment to minimize the filing fee.</p> <p><input checked="" type="checkbox"/> A SECOND or SUBSEQUENT preliminary amendment.</p> <p>14. <input type="checkbox"/> A substitute specification.</p> <p>15. <input type="checkbox"/> A change of power of attorney and/or address letter.</p> <p>16. <input checked="" type="checkbox"/> Other items or information:</p> <ol style="list-style-type: none"> a. a Return Receipt Postcard; b. Form PTO-2038 to cover the filing fee; c. 2 sheets of formal drawings with 3 Figs.; d. copy of International Search Report with English Version; e. marked-up version of Literal Trans. Pages 5, 7 and 9. | | | |
| NOTE: The Priority of German Patent Application 199 27 823.7, filed in the Federal Republic of Germany on June 18, 1999 is claimed under 35 U.S.C. §119. | | | |

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|---------------------------------------------------------------------------------------------------------------------------------------------------------------------------------------------------------------------------------------------------------------|---------------------|---------------------------------------------------------------------------------------------------|--------------------------------------|
| US APPLICATION NO. (if known) 107019376 (not known - To be assigned) | | INTERNATIONAL APPLICATION NO PCT/EP00/05210 | ATTORNEY'S DOCKET NUMBER 4223/PCT |
| 17. <input checked="" type="checkbox"/> The following fees are submitted: | | CALCULATIONS PTO USE ONLY | |
| BASIC NATIONAL FEE (37 CFR 1.492(a)(1)-(5)): | | | |
| Neither international preliminary examination fee (37 CFR 1.482) nor international search fee (37 CFR 1.445(a)(2)) paid to USPTO and International Search Report not prepared by the EPO or JPO \$970.00 | | | |
| International preliminary examination fee (37 CFR 1.482) not paid to USPTO but International Search Report prepared by the EPO or JPO \$890.00 \$840.00 | | | |
| International preliminary examination fee (37 CFR 1.482) not paid to USPTO but international search fee (37 CFR 1.445(a)(2)) paid to USPTO \$769.00 | | | |
| International preliminary examination fee paid to USPTO (37 CFR 1.482) but all claims did not satisfy provisions of PCT Article 33(1)-(4) \$678.00 | | | |
| International preliminary examination fee paid to USPTO (37 CFR 1.482) and all claims satisfied provisions of PCT Article 33(1)-(4) \$26.00 | | | |
| ENTER APPROPRIATE BASIC FEE AMOUNT = | | \$ 890.00 | |
| Surcharge of \$130.00 for furnishing the oath or declaration later than <input type="checkbox"/> 20 <input type="checkbox"/> 30 months from the earliest claimed priority date (37 CFR 1.492(e)). | | \$ 0 | |
| CLAIMS | NUMBER FILED | NUMBER EXTRA | RATE |
| Total claims | 7 - 20 = | 0 | X \$18.00 |
| Independent claims | 2 - 3 = | 0 | X \$78.00 |
| MULTIPLE DEPENDENT CLAIM(S) (if applicable) | | + \$260.00 | |
| TOTAL OF ABOVE CALCULATIONS = | | \$ 890.00 | |
| Reduction of 1/2 for filing by small entity, if applicable. A Small Entity Statement must also be filed (Note 37 CFR 1.9, 1.27, 1.28). | | \$ 0 | |
| SUBTOTAL = | | \$ 890.00 | |
| Processing fee of \$130.00 for furnishing the English translation later than <input type="checkbox"/> 20 <input type="checkbox"/> 30 months from the earliest claimed priority date (37 CFR 1.492(f)). | | \$ 0 | |
| TOTAL NATIONAL FEE = | | \$ 890.00 | |
| Fee for recording the enclosed assignment (37 CFR 1.21(h)). The assignment must be accompanied by an appropriate cover sheet (37 CFR 3.28, 3.31). \$40.00 per property | | + \$ 0 | |
| TOTAL FEES ENCLOSED = | | \$ 890.00 | |
| | | Amount to be: refunded | \$ |
| | | charged | \$ |
| a. <input checked="" type="checkbox"/> Form PTO-2038 (Credit Card Payment Form) A check in the amount of <u>\$ 890.00</u> to cover the above fees is enclosed. | | | |
| b. <input type="checkbox"/> Please charge my Deposit Account No. _____ in the amount of \$_____ to cover the above fees. A duplicate copy of this sheet is enclosed. | | | |
| c. <input checked="" type="checkbox"/> The Commissioner is hereby authorized to charge any additional fees which may be required, or credit any overpayment to Deposit Account No. <u>50-0507</u> . XX A duplicate copy of this sheet is enclosed. | | | |
| NOTE: Where an appropriate time limit under 37 CFR 1.494 or 1.495 has not been met, a petition to revive (37 CFR 1.137(a) or (b)) must be filed and granted to restore the application to pending status. | | | |
| SEND ALL CORRESPONDENCE TO CUSTOMER NO.: 021553 | |  SIGNATURE | |
| USPS EXPRESS MAIL EL 871 050 069 US DECEMBER 18 2001 | | Walter F. Fasse 12/18/01 NAME 36132 REGISTRATION NUMBER | |

10/019336

531 Recd PCT 18 DEC 2001

DOCKET NO.: 4223/PCT

IN THE UNITED STATES PATENT AND TRADEMARK OFFICE

IN THE MATTER OF THE **PCT NATIONAL PHASE PATENT APPLICATION**

OF: Alexander VON GAISBERG et al.

USPS EXPRESS MAIL

EL 871 050 069 US

USSN: TO BE ASSIGNED - NEW

DECEMBER 18 2001

FILED: December 18, 2001

FOR: Electromagnetic Actuator and Method for
Adjusting Said Electromagnetic Actuator

INTERNATIONAL SERIAL NO.: PCT/EP00/05210

INTERNATIONAL FILING DATE: 07. JUNE 2000 (07.06.00)

ASSISTANT COMMISSIONER FOR PATENTS
BOX PCT
WASHINGTON, D. C. 20231

December 18, 2001

FIRST PRELIMINARY AMENDMENT TO MINIMIZE THE FILING FEE

Dear Sir:

In order to minimize the filing fee, please amend the above identified patent application as follows before calculating the filing fee.

Referring to the Literal Translation of International Application
PCT/EP00/05210

In the Claims:

Please cancel claims **4, 5 and 6.**

Claims **1 to 3** and **7** are maintained for calculating the filing fee.

REMARKS:

After calculating the filing fee, please further enter the accompanying Second Preliminary Amendment which introduces new claims **8, 9** and **10** for examination.

Respectfully submitted,

Alexander VON GAISBERG et al.
Applicant

By Walter Fasse

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WFF:ar/4223/PCT
Encls.: postcard

10/019336

531 Rec'd PCT/WF. 18 DEC 2001

DOCKET NO.: 4223/PCT

IN THE UNITED STATES PATENT AND TRADEMARK OFFICE

IN THE MATTER OF THE **PCT NATIONAL PHASE PATENT APPLICATION**

OF: Alexander VON GAISBERG et al.

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ASSISTANT COMMISSIONER FOR PATENTS
BOX PCT
WASHINGTON, D. C. 20231

December 18, 2001

SECOND PRELIMINARY AMENDMENT

Dear Sir:

After calculating the filing fee, but before the first examination, please amend the above identified application as follows.

In the Specification: (Referring to the Literal Translation)

Please delete and replace the paragraph at **page 4, line 12 to page 5, line 13**, to read as follows:

According to the Fig. 1, the actuator according to the invention comprises a push rod or valve stem 4 that is in

force transmitting cooperation with a gas exchange valve 5, an armature 1 secured with the valve stem 4 perpendicularly to the valve stem longitudinal axis, an electromagnet 3 acting as a closing magnet, as well as a further electromagnet 2 acting as an opening magnet, which is arranged spaced apart from the closing magnet 3 in the direction of the valve stem longitudinal axis. The electromagnets 2, 3 respectively comprise an energizing or exciting coil 20 or 30, and pole surfaces lying across from one another. By means of an alternating energization of both electromagnets 2, 3, that is to say the exciting coils 20 or 30, the armature 1 is moved back and forth between the electromagnets 2, 3 along a stroke travel that is limited by the electromagnets 2, 3. A spring arrangement with a first spring 61 acting in the opening direction onto the armature 1 and a second spring 62 acting in the closing direction onto the armature 1 effectuates that the armature 1 is held in a neutral equilibrium position between the electromagnets 2, 3 in the de-energized condition of the exciting coils 20, 30. Furthermore, adjusting or setting means 71, 72 for setting the pre-stressing of the springs 61, 62 are provided. The setting means 71, 72 may, for example, be embodied as disks, which effectuate a compression of the springs 61, 62, and thereby prescribe the pre-stressing of the respective springs 61, 62. They may, however, also be controllably embodied, and enable a stepless variation of the pre-stressing.

Please delete and replace the paragraph at page 6, line 14 to page 7, line 23, to read as follows:

The stroke travel distance I_m of the armature 1, over which the armature 1 travels - the motion of the armature 1 is referred to as flight in the following - is limited due to the prescribed spacing distance between the electromagnets 2, 3. The progressions of the spring forces of the two springs 61, 62, that is to say the forces with which the springs 61, 62 act on the armature 1, are dependent on the armature position I and can be described in connection with spring characteristic curves. In the force versus travel distance diagram of Fig. 2, the spring characteristic curve of the first spring 61 is referenced with F_1 , and the spring characteristic curve of the second spring 62 is referenced with F_2 . During the flight of the armature 1 from the upper end position to the lower end position, that is to say from the armature position 0 to the armature position I_m , the force of the first spring 61 increases at first from a holding value F_{11} to a maximum value F_{13} , which is achieved at the armature position I_x , in order to thereafter fall off to an end value F_{10} lying below the holding value F_{11} , whereby the end value F_{10} is achieved at the armature position I_m , that is to say in connection with the armature 1 lying against the opening magnet 2. In contrast, the spring force of the second spring 62 increases from an end value F_{20} , which is effective in the in the upper end position of the armature 1, monotonously

but non-linearly to a holding value F21, which is achieved in the lower end position of the armature 1. The end values F10, F20 give the pre-stressing of the respective spring 61 or 62; they are adjusted or set in such a manner so that the area A1 under the spring characteristic curve F1 is equal to the area A2 under the spring characteristic curve F2. The areas A1 and A2 in that context correspond to the energy that is stored in the respective spring 61, 62, if these are compressed due to the motion of the armature. The two spring characteristic curves F1, F2 intersect each other at a point that prescribes the energetic center position Ie of the armature 1; this energetic center position Ie, which the armature 1 takes up with de-energized electromagnets 2, 3, generally does not correspond with the geometric center position between the electromagnets 2, 3 in connection with springs with different spring characteristic curves.

Please delete and replace the paragraph at **page 9, lines 8 to 20**, to read as follows:

The energy that is stored in the first spring 61 if the armature 1 is moved from its lower end position to its upper end position, is also measured in the same manner as described above, namely by measuring the progression of the spring force of the first spring 61 that results from the armature motion, and by integration of this progression over the spring travel distance, through which the first

spring 61 is thereby compressed. Next, the energy values that have been determined in this manner are compared with one another, and the pre-stressing of the first spring 61 is adjustingly set in such a manner so that the same energy is stored in the two springs 61, 62, if these are compressed by the stroke travel distance l_m . The actuator is only installed into the internal combustion machine after this adjustment.

In the Claims:

Claims 1 to 3 and 7 are maintained unchanged.

Claims 4, 5 and 6 have previously been cancelled in applicants' First Preliminary Amendment.

Please enter new claims 8, 9 and 10 as follows.

8. (new) Electromagnetic actuator according to claim 1, characterized in that setting means (71, 72) for setting the pre-stressing of the springs (61, 62) are provided.
9. Electromagnetic actuator according to claim 8, characterized in that measuring means for measuring the progressions of the spring forces of the springs (61, 62) are provided.
10. Electromagnetic actuator according to claim 9, characterized in that control means for controlling the

3 setting means in accordance with the measured progressions
4 of the spring forces are provided.

REMARKS:

1) Examination of the present U. S. National Phase Application is to proceed on the basis of claims 1 to 3 and 7 to 10. Claims 8, 9 and 10 are directly based on the translated PCT International Application claims 4, 5 and 6, except for omitting multiple dependencies.

2) A few typographical errors (e.g. in reference numbers) that existed in the PCT International Application text and the literal English translation thereof have been corrected in the present amendment of the specification. A marked-up version of the amended pages of the specification is enclosed to show the amendatory subject matter. No new matter has been introduced. Any further informalities of the literally translated specification and claims will be addressed after receiving the first Office Action.

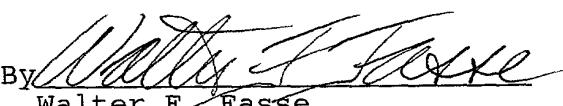
3) It is noted that the International Preliminary Examination Report indicates that at least claims 5 to 7 (i.e. present claims 9, 10 and 7) satisfy all criteria for patentability under the PCT.

4) Favorable consideration and allowance of claims 1 to 3 and 7 to 10 are respectfully requested.

Respectfully submitted,

Alexander VON GAISBERG et al.
Applicant

WFF:ar/4223/PCT
Encls.: postcard, marked-up
version of Lit. Trans. pages
5, 7, 9

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DECEMBER 18 2001

"Marked-up Version"

stroke travel that is limited by the electromagnets 2, 3. A
spring arrangement with a first spring 61 acting in the opening
direction onto the armature 1 and a second spring 62 acting in
the closing direction onto the armature 1 effectuates that the
armature 1 is held in a neutral equilibrium position between the
electromagnets 2, 3 in the de-energized condition of the exciting
coils 20, 30. Furthermore, adjusting or setting means 71, 72 for
setting the pre-stressing of the springs 61, 62 are provided.
The setting means 71, 72 may, for example, be embodied as disks,
which effectuate a compression of the springs [61, 62], and thereby
prescribe the pre-stressing of the respective springs [61, 62].
They may, however, also be controllably embodied, and enable a
stepless variation of the pre-stressing.

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For starting the actuator, one of the electromagnets 2, 3 is energized, that is to say switched on, by applying an exciting voltage to the corresponding exciting coil 20 or 30, or a transient start-up oscillation routine is initiated, by means of which the armature 1 is first set into oscillation by alternating energization of the electromagnets 2, 3, in order to strike against the pole surface of the closing magnet 2 or the pole surface of the opening magnet 3 after a start-up oscillation transient time.

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With a closed gas exchange valve 5, the armature 1 lies against the pole surface of the closing magnet 3 as shown in Fig. 1, and it is held in this position - the upper end position - as long as the closing magnet 3 is energized. In order to open the gas exchange valve 5, the closing magnet 3 is switched off, and next

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a maximum value F13, which is achieved at the armature position IX, in order to thereafter fall off to an end value F10 lying below the holding value F11, whereby the end value F10 is achieved at the armature position Im, that is to say in connection with the armature 1 lying against the opening magnet 2. In contrast, the spring force of the second spring 62 increases from an end value F20, which is effective in the [in the] upper end position of the armature 1, monotonously but non-linearly to a holding value F21, which is achieved in the lower end position of the armature 1. The end values F10, F20 give the pre-stressing of the respective spring 61 or 62; they are adjusted or set in such a manner so that the area A1 under the spring characteristic curve F1 is equal to the area A2 under the spring characteristic curve F2. The areas A1 and A2 in that context correspond to the energy that is stored in the respective spring 61, 62, if these are compressed due to the motion of the armature. The two spring characteristic curves $\underline{F1, F2}$ intersect each other at a point that prescribes the energetic center position Ie of the armature 1; this energetic center position Ie, which the armature 1 takes up with de-energized electromagnets 2, 3, generally does not correspond with the geometric center position between the electromagnets 2, 3 in connection with springs with different spring characteristic curves.

On the one hand, the substantial advantage of the first spring 61, due to the maximum value F13 of its spring characteristic curve F1, is that it is in the position to store so much energy, that the armature 1 will be moved with high velocity during the de-stressing of the first spring 61, which leads to short switch-

corresponding to the stroke travel distance I_m of the armature 1, and the progression of the spring force, which results thereby, is measured section-wise and integrated section-wise over the spring travel distance. The result of this integration 5 corresponds to the energy that is stored in this context in the second spring 62. Thereby, the measurement of the spring force can be carried out by means of a load cell or a measuring gage.

The energy that is stored in the first spring 61 if the armature 1 is moved from its lower end position to its upper end position, 10 is also measured in the same manner as described above, namely by measuring the progression of the spring force of the first spring 61 that results from the armature motion, and by integration of this progression over the spring travel distance, through which the first spring 61 is thereby compressed. Next, the energy values that have been determined in this manner are compared with one another, and the pre-stressing of the first spring 61 is adjustingly set in such a manner so that the same energy 15 is stored in the two springs 61, [61], if these are compressed by the stroke travel distance I_m . The actuator is only installed 20 into the internal combustion machine after this adjustment.

In the present example embodiment, the actuator is adjusted before placing it into operation. Also conceivable, however, are an adjustment during the operation, and an after-adjustment 25 dependent on operating parameters. In this case, the adjusting or setting means are controllably embodied, and the progressions of the spring forces are measured with measuring means, onto which the springs act, for example with pressure sensors, espe-

2/PRTS

Docket # 4223/PCT
INN; Alexander VON GANSBERG
10/019336 et al.

531 Rec'd PCT/PTC 18 DEC 2001

LITERAL TRANSLATION OF PCT INTERNATIONAL APPLICATION
PCT/EP00/05210 FILED ON JUNE 7, 2000

Electromagnetic Actuator and Method for Adjusting Said Electromagnetic Actuator

The invention relates to an electromagnetic actuator according to the preamble of the patent claim 1 and a method for the adjusting of an electromagnetic actuator according to the preamble of the patent claim 6.

An electromagnetic actuator for operating a gas exchange valve in an internal combustion machine is known from the DE 196 31 909 A1. The actuator comprises two electromagnets arranged at a spacing distance relative to one another, and an armature that is in operative connection with the gas exchange valve and that is movable back and forth by magnetic force between the electromagnets against the force of two respectively counteracting springs. The actuator further comprises setting means, with which the position of the armature is set to the geometric center position between the two end positions of the armature in connection with de-energized electromagnets. In this regard, the high dependency of the energy requirement of the actuator on production tolerances is found to be disadvantageous.

Therefore, the invention is based on the object to provide an electromagnetic actuator according to the preamble of the patent claim 1, of which the energy requirement slightly depends on the production tolerances. The invention is further based on the object to provide a method according to the preamble of the

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patent claim 6, through which the dependency of the energy requirement of the actuator on production tolerances is minimized.

The object is achieved in an electromagnetic actuator according to the preamble of the patent claim 1 by the characterizing features of the patent claim 1, and in a method according to the preamble of the patent claim 7 by the characterizing features of the patent claim 7.

Advantageous embodiments and further developments are evident from the dependent claims.

According to the invention, the springs are pre-stressed in such a manner that the same energy will be stored in both springs in connection with a compression of the springs respectively by a spring travel distance prescribed by the limited stroke travel distance of the armature. Hereby one achieves, that the armature, when it is released from its two end positions and freely oscillates, will approach close to the two electromagnets to the same extent. As a result thereof, the influence of production-necessitated tolerances of the components, and especially of the springs, on the oscillating behavior of the armature is reduced.

Moreover, the total energy requirement of the actuator is optimized, because both electromagnets comprise the same current requirement due to the armature approaching equally closely to the two electromagnets. Namely, if the armature would approach more closely to the one electromagnet than to the other during the free oscillation, then the current requirement of the one electromagnetic would drop by a certain amount, but the current

requirement of the other electromagnet would increase by a multiple of this amount, so that also the total energy requirement of the actuator would increase relative to the optimal value.

5 Preferably, at least one of the springs comprises a non-linear spring characteristic, advantageously a characteristic with a maximum value at a position of the armature lying between the electromagnets. Due to the non-linear spring characteristic of one or both of the springs, it is on the one hand ensured that the armature is accelerated with large forces, which has a high switching frequency as a result, and on the other hand one thereby achieves that small forces act in the end positions of the armature, so that also the energy requirement of the actuator for holding the armature in its end positions is small.

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For the adjustment of this electromagnetic actuator, for each spring the course or progression of the spring force is measured, which spring force arises if the respective spring is compressed by a spring travel distance corresponding to the stroke travel distance of the armature. The energy, which is stored in the respective spring due to the compression thereof, is determined from the measured curves or progressions of the spring forces. Next, the pre-stressing of one or both springs is set in such a manner that the same energy is stored in both springs.

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The adjustment of the actuator can be carried out during the manufacturing of the actuator, but an adjustment during the operation is also conceivable, in order to compensate changes of

operating values or parameters, as they may arise, for example, due to temperature effects, wear, or aging.

The invention will be described in greater detail below in connection with an example embodiment, with reference to the Figures, wherein:

Fig. 1 shows an electromagnetic actuator for operating a gas exchange valve in an internal combustion machine,

Fig. 2 shows a first force versus travel distance diagram with spring characteristic curves,

Fig. 3 shows a second force versus travel distance diagram with spring characteristic curves.

According to the Fig. 1, the actuator according to the invention comprises a push rod or valve stem 4 that is in force transmitting cooperation with a gas exchange valve 5, an armature 1 secured with the valve stem 4 perpendicularly to the valve stem longitudinal axis, an electromagnet 3 acting as a closing magnet as well as a further electromagnet 2 acting as an opening magnet, which is arranged spaced apart from the closing magnet 3 in the direction of the valve stem longitudinal axis. The electromagnets 2, 3 respectively comprise an energizing or exciting coil 20 or 30, and pole surfaces lying across from one another. By means of an alternating energization of both electromagnets 2, 3, that is to say the exciting coils 20 or 30, the armature 1 is moved back and forth between the electromagnets 2, 3 along a

stroke travel that is limited by the electromagnets 2, 3. A spring arrangement with a first spring 61 acting in the opening direction onto the armature 1 and a second spring 62 acting in the closing direction onto the armature 1 effectuate that the 5 armature 1 is held in a neutral equilibrium position between the electromagnets 2, 3 in the de-energized condition of the exciting coils 20, 30. Furthermore, adjusting or setting means 71, 72 for setting the pre-stressing of the springs 61, 62 are provided. The setting means 71, 72 may, for example, be embodied as disks, 10 which effectuate a compression of the springs 71, 72, and thereby prescribe the pre-stressing of the respective springs 71, 72. They may, however, also be controllably embodied, and enable a stepless variation of the pre-stressing.

For starting the actuator, one of the electromagnets 2, 3 is energized, that is to say switched on, by applying an exciting voltage to the corresponding exciting coil 20 or 30, or a transient start-up oscillation routine is initiated, by means of which the armature 1 is first set into oscillation by alternating energization of the electromagnets 2, 3, in order to strike 20 against the pole surface of the closing magnet 2 or the pole surface of the opening magnet 3 after a start-up oscillation transient time.

With a closed gas exchange valve 5, the armature 1 lies against the pole surface of the closing magnet 3 as shown in Fig. 1, and 25 it is held in this position - the upper end position - as long as the closing magnet 3 is energized. In order to open the gas exchange valve 5, the closing magnet 3 is switched off, and next

the opening magnet 2 is switched on. The first spring 61 acting
in the opening direction accelerates the armature 1 through or
beyond the rest position. By means of the opening magnet 2 which
is now energized, additional kinetic energy is supplied to the
5 armature 1, so that it reaches the pole surface of the opening
magnet 2 despite possible friction losses, and there - at the
bottom end position which is shown with dashed lines in
Fig. 1 - is held until the switching off of the opening magnet
2. For the renewed closing of the gas exchange valve 5, the
10 opening magnet 2 is switched off and the closing magnet 3 is next
again switched on. The armature 1 is thereby moved by the second
spring 62 to the closing magnet 3 and is held there on its pole
surface.

The stroke travel distance I_m of the armature 1, over which the
armature 1 travels - the motion of the armature 1 is referred
to as flight in the following - is limited due to the prescribed
spacing distance between the electromagnets 2, 3. The progress-
sions of the spring forces of the two springs 61, 62, that is to
say the forces with which the springs 61, 62 act on the armature
1, are dependent on the armature position I and can be described
20 in connection with spring characteristic curves. In the force
versus travel distance diagram of Fig. 2, the spring characteris-
tic curve of the first spring 61 is referenced with F_1 , and the
spring characteristic curve of the second spring 62 is referenced
with F_2 . During the flight of the armature 1 from the upper end
position to the lower end position, that is to say from the
25 armature position 0 to the armature position I_m , the force of the
first spring 61 increases at first from a holding value F_{11} to

a maximum value F_{13} , which is achieved at the armature position I_x , in order to thereafter fall off to an end value F_{10} lying below the holding value F_{11} , whereby the end value F_{10} is achieved at the armature position I_m , that is to say in connection with the armature 1 lying against the opening magnet 2. In contrast, the spring force of the second spring 62 increases from an end value F_{20} , which is effective in the upper end position of the armature 1, monotonously but non-linearly to a holding value F_{21} , which is achieved in the lower end position of the armature 1. The end values F_{10} , F_{20} give the pre-stressing of the respective spring 61 or 62; they are adjusted or set in such a manner so that the area A_1 under the spring characteristic curve F_1 is equal to the area A_2 under the spring characteristic curve F_2 . The areas A_1 and A_2 in that context correspond to the energy that is stored in the respective spring 61, 62, if these are compressed due to the motion of the armature. The two spring characteristic curves 61, 62 intersect each other at a point that prescribes the energetic center position I_e of the armature 1; this energetic center position I_e , which the armature 1 takes up with de-energized electromagnets 2, 3, generally does not correspond with the geometric center position between the electromagnets 2, 3 in connection with springs with different spring characteristic curves.

On the one hand, the substantial advantage of the first spring 61, due to the maximum value F_{13} of its spring characteristic curve F_1 , is that it is in the position to store so much energy, that the armature 1 will be moved with high velocity during the de-stressing of the first spring 61, which leads to short switch-

ing times, despite the small holding value F_{11} . Due to the small holding value F_{11} , on the other hand, the current requirement for holding the armature 1 in its upper end position, and therewith the energy requirement of the actuator, is small.

5 In the force versus travel distance diagram according to Fig. 3, the spring characteristic curve F_2 of the second spring 62, with an increasing spacing distance I between armature 1 and closing magnet 2, comprises at first a decreasing progression, then an increasing progression, and thereafter again a decreasing progression. The areas A_1 , A_2 under the spring characteristic curves F_1 , F_2 of the springs 61, 62 are once again equally large. For these spring characteristic curves F_1 , F_2 it is shown to be advantageous, that the difference ΔF between the two spring characteristic curves F_1 , F_2 , that is to say the resulting force acting on the armature 1, is large for a large range of the spacing distance I between the armature 1 and closing magnet 3. As a result of that, the gas exchange valve 5 may also be opened against a combustion chamber internal pressure, that is to say the energy requirement of the opening magnet 2 is small due to 10
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20 the high resulting force ΔF that is effective during the opening process.

The adjustment of the actuator is carried out before the installation of the actuator in the internal combustion machine. Thereby, first the pre-stressing of the second spring 62 is 25
adjustingly set to the end value F_{20} , at which a secure or reliable closing of the gas exchange valve 5 is ensured. Next, the second spring 62 is compressed by the spring travel distance

corresponding to the stroke travel distance I_m of the armature 1, and the progression of the spring force, which results thereby, is measured section-wise and integrated section-wise over the spring travel distance. The result of this integration 5 corresponds to the energy that is stored in this context in the second spring 62. Thereby, the measurement of the spring force can be carried out by means of a load cell or a measuring gage.

The energy that is stored in the first spring 61 if the armature 1 is moved from its lower end position to its upper end position, 10 is also measured in the same manner as described above, namely by measuring the progression of the spring force of the first spring 61 that results from the armature motion, and by integration of this progression over the spring travel distance, through which the first spring 61 is thereby compressed. Next, the energy values that have been determined in this manner are compared with one another, and the pre-stressing of the first spring 61 is adjustingly set in such a manner so that the same energy 20 is stored in the two springs 61, 61, if these are compressed by the stroke travel distance I_m . The actuator is only installed into the internal combustion machine after this adjustment.

In the present example embodiment, the actuator is adjusted before placing it into operation. Also conceivable, however, are an adjustment during the operation, and an after-adjustment dependent on operating parameters. In this case, the adjusting 25 or setting means are controllably embodied, and the progressions of the spring forces are measured with measuring means, onto which the springs act, for example with pressure sensors, espe-

cially with piezocrystals. The adjusting or setting means are then controlled by control means, dependent on the measured spring forces, in such a manner so that the same energy is stored in both springs in connection with the maximum compression of the 5 springs 61, 62 that is possible during the operation.

Patent Claims:

1. Electromagnetic actuator with two electromagnets arranged at a spacing distance relative to one another, and an armature (1) that is movable back and forth along a stroke travel distance (I_m) between the electromagnets (2, 3) against the force of two springs (61, 62) acting against each other, characterized in that the springs (61, 62) are pre-stressed in such a manner, so that the same energy (A_1 , A_2) is stored in both springs (61, 62) in connection with a compression of the springs (61, 62) that is prescribed by the stroke travel distance (I_m) of the armature (1).
2. Electromagnetic actuator according to claim 1, characterized in that at least one of the springs (61, 62) comprises a non-linear spring characteristic curve (F_1).
3. Electromagnetic actuator according to claim 2, characterized in that the spring characteristic curve (F_1) of at least one of the springs (61, 62) comprises a maximum value (F_{13}) at a position (I_x) of the armature (1) spaced away from the two electromagnets (2, 3).
4. Electromagnetic actuator according to one of the claims 1 to 3, characterized in that setting means (71, 72) for setting the pre-stressing of the springs (61, 62) are provided.

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7. Method for the adjusting of an electromagnetic actuator with two electromagnets (2, 3) arranged at a spacing distance relative to each other, and an armature (1) movable back and forth along a stroke travel distance between the electromagnets (2, 3) against the force of two springs (61, 62) acting against one another, characterized in that, for each spring (61, 62) the progression (F1, F2) of the spring force is measured, which results if the respective spring (61, 62) is compressed by a spring travel distance corresponding to the stroke travel distance (Im) of the armature (1), that in connection with the measured progressions (F1, F2) of the spring forces, the energy (A1, A2) is determined, which is stored in the respective spring (61, 62) due to the compression thereof, and that the pre-stressing (F10, F20) of one or both springs (61, 62) is set in such a manner so that the same energy (A1, A2) is stored in both springs (61, 62).

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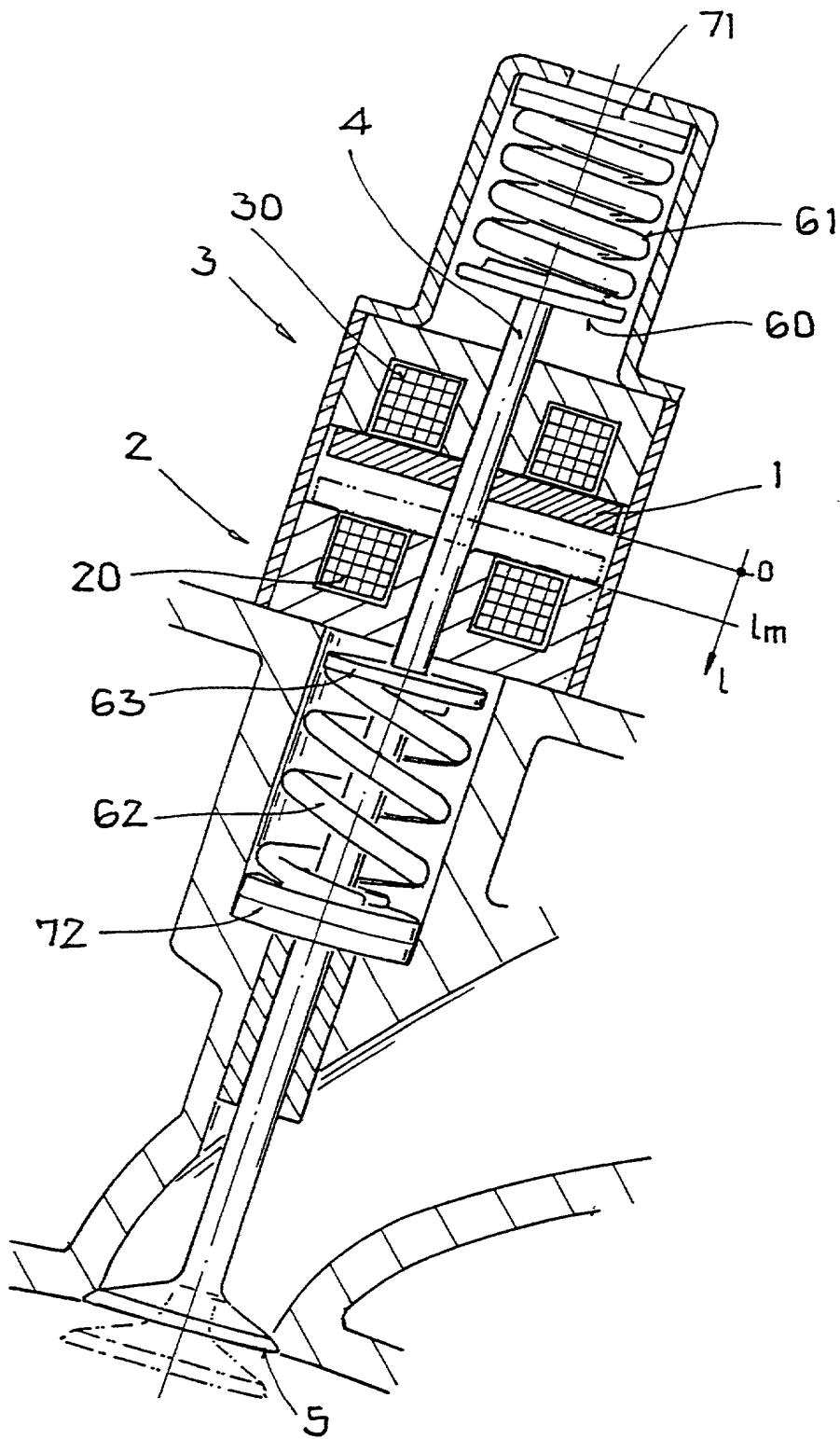
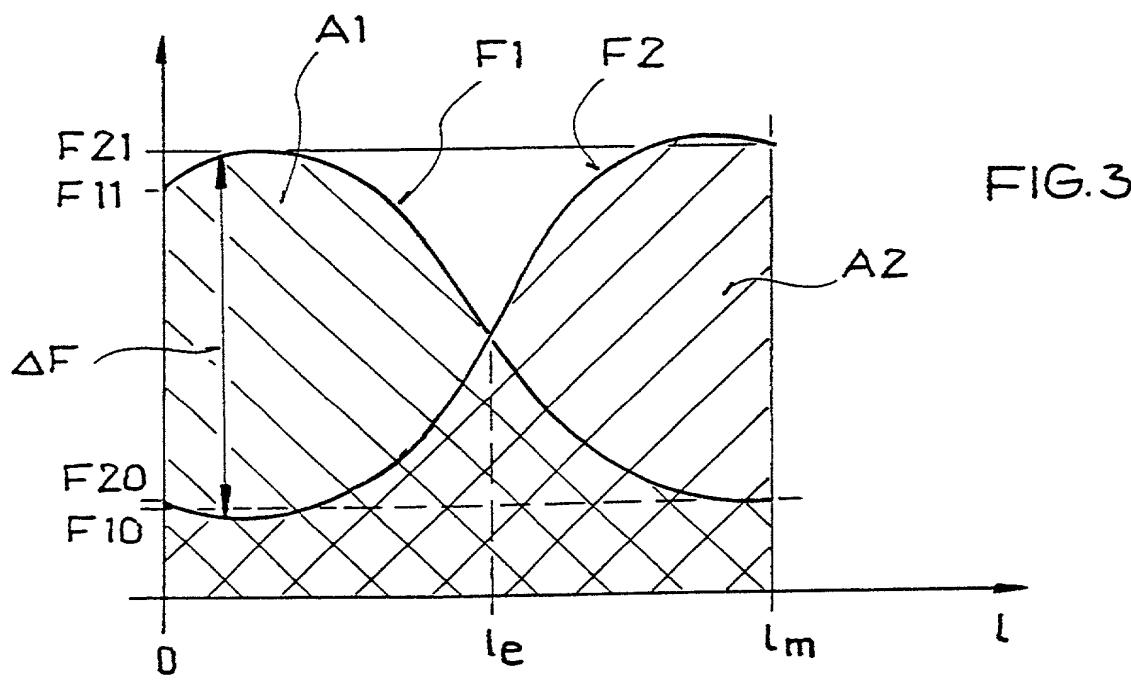
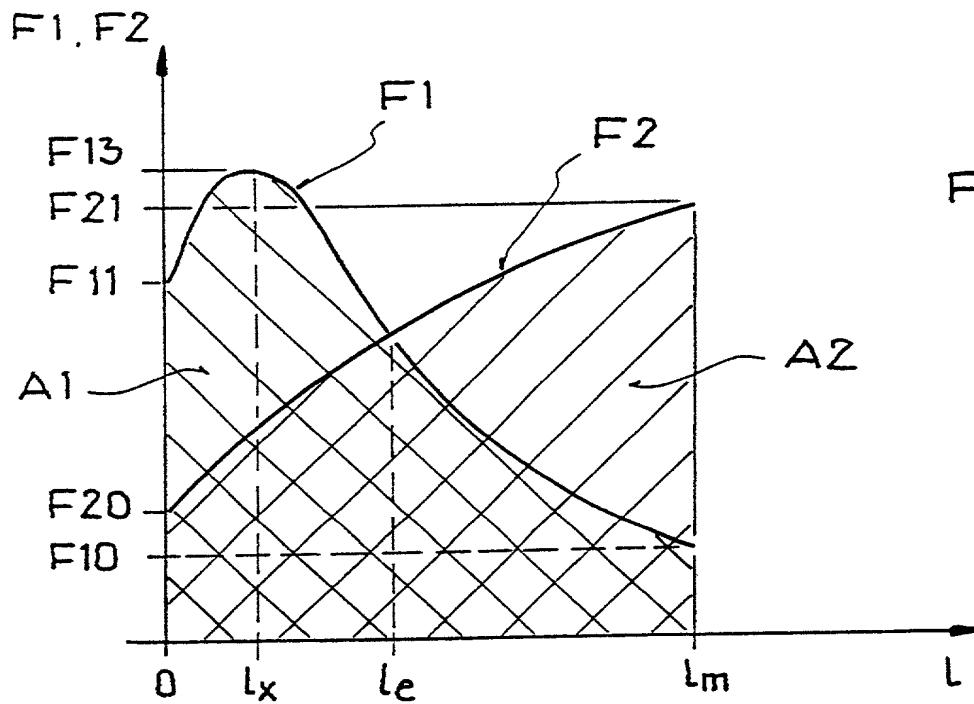


FIG. 1

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**DECLARATION FOR UTILITY OR
DESIGN
PATENT APPLICATION
(37 CFR 1.63)
WITH POWER OF ATTORNEY**

Declaration Submitted with Initial Filing Declaration Submitted after Initial Filing (surchARGE (37 CFR 1.16 (e)) required)

| | |
|--------------------------|------------------------|
| Attorney Docket Number | 4223 |
| First Named Inventor | Alexander von GAISBERG |
| COMPLETE IF KNOWN | |
| Application Number | / |
| Filing Date | |
| Group Art Unit | |
| Examiner Name | |

As a below named inventor, I hereby declare that:

My residence, mailing address, and citizenship are as stated below next to my name.

I believe I am the original, first and sole inventor (if only one name is listed below) or an original, first and joint inventor (if plural names are listed below) of the subject matter which is claimed and for which a patent is sought on the invention entitled:

ELECTROMAGNETIC ACTUATOR AND METHOD FOR ADJUSTING SAID ELECTROMAGNETIC ACTUATOR

(Title of the Invention)

the specification of which

is attached hereto
OR
 was filed on (MM/DD/YYYY) as United States Application Number or PCT International

Application Number and was amended on (MM/DD/YYYY) (if applicable).

I hereby state that I have reviewed and understand the contents of the above identified specification, including the claims, as amended by any amendment specifically referred to above.

I acknowledge the duty to disclose information which is material to patentability as defined in 37 CFR 1.56, including for continuation-in-part applications, material information which became available between the filing date of the prior application and the national or PCT international filing date of the continuation-in-part application.

I hereby claim foreign priority benefits under 35 U.S.C. 119(a)-(d) or 365(b) of any foreign application(s) for patent or inventor's certificate, or 365(a) of any PCT international application which designated at least one country other than the United States of America, listed below and have also identified below, by checking the box, any foreign application for patent or inventor's certificate, or any PCT international application having a filing date before that of the application on which priority is claimed.

| Prior Foreign Application Number(s) | Country | Foreign Filing Date (MM/DD/YYYY) | Priority Not Claimed | Certified Copy Attached? YES | Certified Copy Attached? NO |
|-------------------------------------|----------------------|----------------------------------|----------------------------------------------------------------------------------|----------------------------------------------------------------------------------|----------------------------------------------------------------------------------|
| 199 27 823.7 | Fed. Rep. of Germany | 06/18/1999 | <input type="checkbox"/> <input type="checkbox"/> <input type="checkbox"/> | <input type="checkbox"/> <input type="checkbox"/> <input type="checkbox"/> | <input type="checkbox"/> <input type="checkbox"/> <input type="checkbox"/> |

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| U.S. Parent Application or PCT Parent Number | Parent Filing Date (MM/DD/YYYY) | Parent Patent Number (if applicable) |
|----------------------------------------------|---------------------------------|-----------------------------------------|
| | | |

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Inventor's Signature A. Gaisberg Date 18.08.01

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Additional inventors are being named on the _____ supplemental Additional Inventor(s) sheet(s) PTO/SB/02A attached hereto.

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